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# A means to estimate thermal and kinetic parameters of coal dust layer from hot surface ignition tests

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#### ABSTRACT

A method to estimate thermal and kinetic parameters of Pittsburgh seam coal subject to thermal runaway is presented using the standard ASTM E 2021 hot surface ignition test apparatus. Parameters include thermal conductivity (k), activation energy (E), coupled term (QA) of heat of reaction (Q) and pre-exponential factor (A) which are required, but rarely known input values to determine the thermal runaway propensity of a dust material. Four different dust layer thicknesses: 6.4, 12.7, 19.1 and 25.4 mm, are tested, and among them, a single steady state dust layer temperature profile of 12.7 mm thick dust layer is used to estimate k, E and OA. k is calculated by equating heat flux from the hot surface layer and heat loss rate on the boundary assuming negligible heat generation in the coal dust layer at a low hot surface temperature. E and QA are calculated by optimizing a numerically estimated steady state dust layer temperature distribution to the experimentally obtained temperature profile of a 12.7 mm thick dust layer. Two unknowns, E and QA, are reduced to one from the correlation of E and QA obtained at criticality of thermal runaway. The estimated k is 0.1 W/mK matching the previously reported value. E ranges from 61.7 to 83.1 kJ/mol, and the corresponding QA ranges from  $1.7 \times 10^9$  to  $4.8 \times 10^{11}$  J/kg s. The mean values of E (72.4 kJ/mol) and QA (2.8 × 10<sup>10</sup> J/kg s) are used to predict the critical hot surface temperatures for other thicknesses, and good agreement is observed between measured and experimental values. Also, the estimated E and QA ranges match the corresponding ranges calculated from the multiple tests method and values reported in previous research.

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#### 1. Introduction

Thermal runaway, also described as supercritical self-heating or spontaneous ignition, has been considered as a serious hazard in many industrial processes and applications such as bulk coal stockpiles [1–3], nickel–cadmium accumulators [4], and dust material deposits on a heated surface [5]. Ignition in dust deposits by thermal runaway can also lead to subsequent dust explosions [6].

Various methods and techniques have been developed to evaluate the propensity of thermal runaway of a material: hot plate test, oven-basket test, thermal analysis test, etc. [7]. Since thermal runaway can occur in various circumstances, each test method has its own merits in application. The hot plate test is specifically designed to evaluate the thermal runaway hazard of a granular material up to a couple of centimeters, a realistic thickness in many industrial environments. The bottom surface of the dust layer is exposed to a hot plate while the top surface is cooled in ambient air. Despite being a relatively short and easy test procedure and resembling actual hazardous conditions closer than the other test methods in terms of sample amount and configuration, hot plate test has been considered just an approximate screening method on the basis of 'go/no go' criteria [8,9]. The oven-basket test, historically the most common test method, represents thermal runaway of granular materials surrounded by constant temperature. This method requires a wire mesh basket, usually cubical in shape, which contains the test material to be placed in an oven at a high temperature. Due to small sample dimensions compared to a real storage size, a high oven temperature is required to cause thermal runaway. The test results are then extrapolated to assess the hazard of a realistic storage stockpile. A review of the oven test and its application to a realistic scenario is given by Jones [10,11]. Thermal analysis tests such as thermogravimetric analysis (TGA) and differential scanning calorimetry (DSC) can measure the critical decomposition temperatures and heat energy produced by the chemical reactions. In these two tests, heat transfer phenomena are of less concern relative to chemical reaction [12] due to a tiny amount of test sample.

Parameters required for determining the thermal runaway hazard of a dust material including thermal conductivity, total heat transfer coefficient, activation energy, and pre-exponential factor in an Arrhenius equation are rarely known or are test environment

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specific. Some of these values can be obtained from other test methods such as TGA and DSC, and multiple hot plate tests with different dust layer thicknesses. The objectives of this study are to estimate these parameters from the hot plate test with a single thickness as compared to the multiple tests method by optimizing a numerical solution to match the experimental temperature distribution in the dust layer. Pittsburgh seam coal dust, one of the benchmark test materials of ASTM E 2021 [13], is used as the test material.

#### 2. Background

The thermal runaway theory by Semenov [14] assumes constant temperature distribution throughout a reaction zone with heat loss at the boundary. However, Semenov's theory is only applicable to cases such as a well-stirred gas mixture due to the assumption of no thermal resistance in the reaction zone. Frank-Kamenetskii [15] adopted temperature distribution in the reaction zone, but assumed no heat loss on the boundary. Limitations of each of these cases were overcome by Thomas and Bowes [18] taking into account temperature distribution both in a reaction zone and heat loss on the boundaries [16].

Thermal runaway implies a sudden temperature increase due to thermal imbalance between heat generation rate and loss rate. Heat generation rate is known to follow the Arrhenius equation as an exponential function of temperature, and heat loss rate can be represented as a linear function of temperature as shown in Fig. 1(A). For an asymmetrically heated dust layer, heat transferred from the hot surface increases the dust layer temperature and consequently leads to a higher heat generation rate. Heat generated by exothermic reaction at the elevated temperature in the layer competes with the heat loss by convection and radiation at the top surface. Therefore, for a given material, layer thickness, and ambient environment, the hot surface temperature is the variable which determines the occurrence of either a thermal balance or thermal runaway. In Fig. 1(A), if the hot plate temperature  $(T_p)$  is set and remains at  $T_{p3}$  where heat generation rate in the dust layer is equal to the loss rate at the boundary, dust layer temperature will remain in steady state. However, any slight increase of heat generation rate can lead to thermal runaway. If  $T_{\rm p}$  is set and remains lower than  $T_{\rm p3}$  such as at  $T_{p2}$  where heat generation rate in the dust layer is higher than heat loss rate, the dust layer temperature will increase up to the point A where a thermal balance exists. A perturbation can cause temperature to increase beyond the point A, but the higher heat loss rate between  $T_{p2}$  and  $T_{p3}$  will direct it back to the steady state point A. Another thermal balance point C can be seen to be unstable since below point C heat loss rate is higher than heat generation rate, which yields temperature drop, and above point C thermal runaway occurs. In case of heating process with starting temperature

lower than  $T_{p3}$ , point A is the only thermal balance point. As hot surface temperature increases from  $T_{p2}$  to  $T_{p3}$ , point C decreases and merges with point A resulting in point B. If  $T_p$  is set and remains at  $T_{p4}$  which is just above  $T_{p3}$ , dust layer temperature continuously increases and reaches thermal runaway.

Presumed temperature distributions in an asymmetrically heated dust layer with thickness 2r are shown in Fig. 1(B). A dust layer comes into contact with a hot surface at x = 0. Line 1 represents linear temperature distribution of an inert material without internal heat generation. The slope is determined by the heat loss rate on the top boundary at x = 2r and thermal conductivity with a given  $T_p$ at  $T_{p1}$ . Curve 2 represents steady state condition at a low hot plate temperature with relatively small amount of heat generation in the dust layer. Curve 3 represents the maximum steady state condition of a dust layer. The maximum layer temperature  $(T_m)$  is observed at  $x_{\rm m}$  very close to the hot surface. Curve 4 represents a transient temperature profile of thermal runaway. Higher oxygen concentration near the open boundary causes more reaction, and consequently, higher temperature than the lower area of the dust layer. The layer ignition temperature (LIT) or the minimum hot plate temperature for thermal runaway exists at  $T_{p4}$ . This is the main concern in most cases, and can be derived from the analytical solution of  $T_{p3}$ , since thermal runaway is theoretically expected to occur just above  $T_{p3}$ .

The analytical solution for  $T_{p3}$ , the maximum hot plate temperature for the dust layer to remain in steady state, is available from Thomas's thermal runaway model [17] with the assumption of negligible reactant depletion [18,19]. Assuming the dust layer as an infinite slab, a one dimensional steady state heat conduction equation can be written as,

$$k\frac{\partial^2 T}{\partial x^2} = -\rho QAe^{-E/RT},\tag{1}$$

where k=thermal conductivity of dust layer (W/mK), T=temperature (K),  $\rho$ =density (kg/m<sup>3</sup>), Q=heat of reaction (J/kg), and A=Arrhenius pre-exponential factor (1/s). Q and A can be treated as one combined term (QA) for mathematical convenience. In the exponential part, E=the activation energy (J/mol) and R=the universal gas constant (=8.314 J/mol K).

Boundary conditions of constant temperature at the bottom surface and Newtonian cooling on the top surface are,

$$T = T_{\rm p} \quad \text{at } x = 0, \tag{2a}$$

$$-k\frac{\mathrm{d}T}{\mathrm{d}x} = h_{\mathrm{t}}(T_{\mathrm{s}} - T_{\mathrm{a}}) \quad \mathrm{at} \ x = 2r, \tag{2b}$$

where  $h_t = h_c + h_r = \text{total heat transfer coefficient (W/m<sup>2</sup> K) accounting for convective (<math>h_c$ ) and radiant ( $h_r$ ) heat transfer.  $T_s = \text{top surface temperature (K)}$ ,  $T_a = \text{ambient temperature (K)}$ .



Fig. 1. Thermal runaway concept (A) and temperature distributions in asymmetrically heated dust layer (B).



**Fig. 2.**  $\delta_c$  as a function of Bi,  $\theta_a$ ,  $\theta_m$ , and  $z_m$  (solid lines) and approximate  $\delta_c$  as a function of corresponding  $\theta_a$  only (dotted lines) for asymmetrically heated slab. The values of  $z_m$  and  $\theta_m$  are selected from Bowes [17].

Eq. (1) can be simplified by non-dimensionalizing the terms for a steady state condition.

$$\frac{\partial^2 \theta}{\partial z^2} = -\delta e^{\theta}, \tag{3a}$$

where

$$\theta = \frac{E}{RT_p^2}(T - T_p)$$
(3b)

and

$$\delta = \frac{E}{RT_p^2} \frac{r^2 \rho QA}{k} e^{(-E/RT_p)},$$
(3c)

 $\theta$ , z = x/r, and  $\delta$  are dimensionless parameters for temperature, distance from a hot surface, and heat generation rate term, respectively, r = Half of the dust layer thickness (m).

Boundary conditions in Eq. (2a) and (2b) become,

$$\theta = 0 \quad \text{at } z = 0,$$
 (4a)

$$-\frac{d\theta}{dz} = Bi(\theta_{\rm s} - \theta_{\rm a}) \quad \text{at } z = 2, \tag{4b}$$

where  $Bi = h_t r/k$  = Biot number.

The non-dimensional maximum heat generation rate for steady state condition,  $\delta_c$ , represented by curve 3 in Fig. 1(B) can be calculated as a function of Bi,  $\theta_a$ ,  $\theta_m$ , and  $z_m$  substituting  $T_p$  with  $T_{p3}$ . Exact solution of  $\delta_c$  is shown in Fig. 2 as solid lines. The values of  $z_m$  and  $\theta_m$  in Fig. 2 are chosen from Bowes [17].

 $\theta_{\rm m}$  and  $z_{\rm m}$  are the maximum layer temperature and its location in the dust layer. Since the maximum temperature in the dust layer occurs almost at the hot surface which yields another boundary

Properties of Pittsburgh seam coal.

condition,  $d\theta/dz = 0$  at z=0,  $\delta_c$  can be approximated without  $z_m$  and subsequent  $\theta_m$  terms. The approximate solution for  $\delta_c$  becomes,

$$\delta_{\rm c} \approx \frac{1}{2} \left( \frac{Bi}{1+2Bi} \right)^2 (1.4 - \theta_{\rm a})^2. \tag{5}$$

Fig. 2 shows the approximate solution as dotted lines.

Fig. 2 or Eq. (5) is used to determine whether thermal runaway is expected to occur for a given material.  $\delta_c$  obtained from Fig. 2 or Eq. (5) is compared to  $\delta$  calculated from Eq. (3c).  $\delta_c$  requires *Bi* and  $\theta_a$ . *Bi* can be calculated if thermal conductivity (*k*), total heat transfer coefficient (*h*<sub>t</sub>) are available, and the dust layer thickness (2r) is measurable. Then, given a hot plate temperature (*T*<sub>p</sub>),  $\theta_a$  is calculated with activation energy (*E*) and ambient temperature (*T*<sub>a</sub>).  $\delta$  is calculated from Eq. (3c). In addition to the parameter values used for  $\delta_c$  calculation, coupled pre-exponential term (*QA*) is required for Eq. (3c). If the condition,  $\delta < \delta_c$  holds, the material is safe from a thermal runaway hazard. In this comparison procedure, *k*, *E*, and *QA* are required input in addition to the measurable dust layer thickness, density, and environment specific total heat transfer coefficient.

For Pittsburgh seam coal, these parameter values can vary with the test method, concerned temperature range, coal type, etc. Even for a given coal from the same mine field, variations exist in thermal properties [20]. Previous studies of thermal and kinetic parameters of Pittsburgh seam coal are shown in Table 1.

#### 3. Experiments

Hot surface ignition tests of Pittsburgh seam coal dust layer with four different thicknesses: 6.4, 12.7, 19.1, and 25.4 mm, have been conducted. Two thermocouples for 6.4 mm and three thermocouples for the rest of the thicknesses were positioned at different heights in each dust layer to measure the temperature profile in the dust. Air flow in the test environment has been known to affect the LIT [29], current test results are from a quiescent environment with average air velocity of less than 0.01 m/s measured for half an hour and with ambient temperature  $(T_a)$  at 22 °C. Test results from 12.7 mm thickness were used to provide base line data for numerical calculation to estimate thermal conductivity and kinetic parameter values. The values of kinetic parameters obtained from a single 12.7 mm thick dust layer were then used to predict critical hot surface temperatures  $(T_{p3})$  for the other layer thicknesses. Predicted  $T_{p3}$  were compared to the expected range of  $T_{p3}$  from the experiment. Test results from the other thicknesses were also used for the estimation of E and QA from the multiple tests method. Details about the multiple tests method are included in Appendix A.

#### 3.1. Experimental set-up

The fundamental set-up and procedures of the dust layer test are similar to the ASTM E 2021[13] and Mirron and Lazzara's tests [8]. The basic test arrangement is shown in Fig. 3.

A circular aluminum disc (A) 25.4 mm in thickness and 203 mm in diameter was used as the hot surface. Disc A was thoroughly attached to the electrically heated plate (F), ROPH-144, Omega

Investigators	E(kJ/mol)	Q (J/kg)	A (1/s)	<i>k</i> (W/mK)	$ ho  (kg/m^3)$	Technique
Howard and Essenhigh [21] Stone et al. [22] Anthony et al. [23] Kobayashi et al. [24] Yuan and Smith [25] Singer and Tye [26]	115.9 102~114 49.4 104.6 89.9		$\begin{array}{c} 4.92 \times 10^4 \\ 1.1 \times 10^5, \sim \!$	0.2 0.19-0.21	1300 1060–1297	Flame furnace Fluidized bed Electric grid Entrained flow Comparative slab
Reddy et al. [28]	65.4	QA = 3.04 $ imes$ 10 <sup>9</sup>		0.27 0.1	492	Hot plate



Fig. 3. Test apparatus arrangement for hot surface dust layer test: cross-sectional and top view with aluminum plate (A), insulation (B), dust layer thermocouples (C), steel ring (D), hot surface thermocouple (E) and hot plate (F) connected to power supply.

engineering, with thermally conductive cement, Omegabond 700, Omega engineering, to provide uniform temperature distribution on the aluminum plate surface. The perimeter wall was also wrapped by a 5 mm thick insulating material (B), ceramic paper 390. Cotronic Co., to minimize the heat loss to the environment. Hot surface temperature was measured by a Type K surface temperature thermocouple (E), CO1-K, Omega engineering, 35 mm inside from the plate edge. Thermal contact between the thermocouple and the hot surface was maintained by a metal clip attached to the plate frame. A temperature controller, CN 8592, Omega engineering, and solid state relay, SSRL240DC25, Omega engineering, were used to control and maintain the hot surface temperature at a steady value throughout any given test. Three stainless steel rings (D) of 102 mm ID, 114 mm OD: 6.4, 12.7 and 25.4 mm in height, respectively, were used to contain the dust layer. Two slots were located at diametrically opposite points on the rim to accommodate thermocouple wires. Type-K, bare thermocouples (C) with bead size of 0.38 mm OD, Omega engineering, were placed throughout the dust layer. Locations of the thermocouples varied with the thickness of the ring, i.e. the thickness of the dust layer. Thermocouples were positioned at 2 and 4 mm in height from the hot surface for 6.4 mm thick ring; 4, 7 and 10 mm for 12.7 mm thick ring; 3, 11, and 16 mm for 19.1 mm thick ring; 6, 12, 18 mm for 25.4 mm thick ring. The 19.1 mm thick ring was made up by stacking the 12.7 mm thick ring on top of the 6.4 mm thick ring. Temperatures of the dust layer were measured and recorded by National Instrument data acquisition system with an additional temperature recording device, HH506RA, Omega engineering, to confirm the temperatures.

Temperatures on the hot aluminum plate set at 250 °C showed very good uniformity. Temperatures were measured at 12 points on the hot surface, uniformly spaced, 6 points each and on two lines normal to each other. Except one point at which temperature was 249 °C, the other 11 points reported 250 °C. The thermocouples for both surface and dust layer temperature have an error of  $\pm 1.1$  °C and the temperature controller, CN8592, has an error range of  $\pm 1$  °C prescribed by the manufacturer. NI data acquisition unit has shown  $\pm 1$  °C temperature fluctuation throughout tests. The inherent uncertainty of test apparatus is within an acceptable range since tests were conducted with either 5 or 10 °C resolution with respect to the layer ignition temperature.

Pittsburgh seam coal dust was provided by NIOSH Pittsburgh research laboratory. Its particle distribution with  $D_{50,vol}$  of 48  $\mu$ m is shown in Fig. 4 as reported by NIOSH. The measured average bulk density was 532 kg/m<sup>3</sup>.

#### 3.2. Experimental procedure

The steel ring was positioned in the center of the aluminum plate and adjusted to fit the slots with thermocouple wires. Bare thermocouples were located at the desired heights through the slots. Since direct contact of thermocouple wire to any metal may result in unstable and inexact temperature reading, the thermocouples were separated from the slots' surface by locating a piece of Cotronics' insulating material between them. After the thermocouples were positioned, the hot plate was turned on, and the desired temperature was set in the temperature controller. Once the hot plate temperature was stabilized, the steel ring was gently filled with a pre-measured amount of coal dust. The surface of the layer was evenly leveled with a flat iron ruler, and coal dust on the aluminum plate was removed. This procedure was carried out carefully so that the location of thermocouples in the dust layer was not disturbed. Tests were run until either the layer temperature reached a steady state for no less than 30 min or clear thermal runaway was observed. If thermal runaway did not occur at the pre-set hot plate temperature  $(T_p)$ , it was increased by 10 °C until thermal runaway was observed. Resolution of 5 °C was adopted for 12.7 mm thick dust layer, the main data provider for numerical estimation, to reduce the error attributed to the test resolution. Fresh coal samples were used for each test.

#### 3.3. Experimental results

Temperatures at 4, 7, and 10 mm heights of 12.7 mm thick layer are shown in Fig. 5(A) for both with and without thermal runaway at hot plate temperatures of 215 and 210 °C. Temperature profiles in the dust layer at specific times: 500, 2000, 2500, 3250 and 3750 s, in Fig. 5(B) are connected to Fig. 5(A) and referred to as a, b, c, d,



Fig. 4. Size distribution of Pittsburgh seam coal dust.



**Fig. 5.** Pittsburgh seam coal dust layer temperatures for a 12.7 mm layer thickness (A) and temperature distributions in the dust layer at corresponding times and inert consideration of coal dust layer (B). In (B), dotted connection line and square marker for  $T_p$  of 210 °C, solid connection line and round maker for  $T_p$  of 215 °C, and asterisk marker for inert case at  $T_p$  of 210 °C.

and e, respectively. Times were selected to highlight aspects of the thermal runaway theory discussed earlier (Fig. 1B). Line (a) represents a typical temperature in the transient period where heat flux from the hot plate increases the dust layer temperature with both a  $T_p$  value of 210 and 215 °C. Between (b) and (c), a transient temperature plateau is shown with a  $T_p$  value of 215 °C as is the result of  $T_p$  of 210 °C. Significant temperature rise during thermal runaway is observed over the period from line (c) to (e) with  $T_p$  of 215 °C as opposed to  $T_p$  of 210 °C with which temperatures slowly decrease and reach steady state.

The presumed temperature distribution in Fig. 1(B) shows good agreement with experimental results in Fig. 5(B).  $T_p$  of 215 °C at which thermal runaway is observed can be interpreted as  $T_{p4}$ , and  $T_p$  of 210 °C can be considered  $T_{p2}$  being located very near to  $T_{p3}$  in Fig. 1(B). Therefore,  $T_{p3}$ , the maximum hot surface temperature for steady state exists between 210 and 215 °C. Dust layer temperature with  $T_p$  of 210 °C at (e) in Fig. 5 can be considered point A in Fig. 1(A) where heat generation and loss rate are balanced. Temperature profile equivalent to the maximum steady state point B in Fig. 1(A) is not explicitly shown, but exists between (b) and (c) in Fig. 5(B). The maximum layer temperature with  $T_p$  of 210 °C is closest to the point B in Fig. 1(A).

Temperatures at all measured points in between (b) and (c) are higher than those of the inert case as shown in Fig. 5(B). This can

be explained by the role of heat generation rate in the dust layer which the inert case does not take into account. The amount of temperature rise depends on the parameter values in the heat generation rate term. The non-dimensional heat generation rate term,  $\delta$  in Eq. (3c), shows the associated parameters affecting the temperatures in the dust layer and the relationship of each parameter to one another.

Thermal runaway was observed at  $T_p$  of 250, 215, 200, and 190 °C for 6.4, 12.7, 19.1, and 25.4 mm thick Pittsburgh seam coal dust layers, respectively, as shown in Fig. 6. All temperature profiles in Fig. 6 were obtained from a single test. For 12.7 mm dust layer thickness when thermal runaway did not occur, the maximum dust layer temperature of 207 °C was recorded at 4 mm height, which is very close to  $T_p$ , but did not surpass it. The assumption for the Eq. (5),  $d\theta/dz = 0$  at z = 0, is in agreement with this result. Time to reach steady state increased as the dust layer thickness increased.

Temperature crossover where the temperature of higher location in the dust layer exceeded those of lower locations was observed except for the 6.4 mm thick dust layer case. The temperature inversion may be caused by the higher oxygen diffusion near the surface facilitating more exothermic reaction in upper region of dust layer. However, oxygen concentration when thermal runaway does not occur is not a limiting factor of exothermic reaction rate [29]. Therefore, it does not have a significant effect on the steady state temperature profile in the dust layer and the following parameter estimation.

#### 4. Parameter estimation

The hot surface ignition of a dust layer is a complex problem since factors such as particle size, contaminants, moisture, etc. are known to influence the minimum ignition temperature. Different contents and impurities of a coal can be roughly, but more often. summarized by a coal rank which also indicates the level of energy. However, it is reported that regardless of coal rank, critical temperature of coal particles with 0.06 mm diameter is about 400 K in oven-basket test [30], meaning that naturally inherent varying contents in coals may not play a critical role in thermal runaway phenomena. Therefore, in this study, Pittsburgh seam coal dust layer is assumed to be a homogenous material. A single 12.7 mm thick dust layer provides test results to estimate thermal conductivity, activation energy, and the coupled pre-exponential term. From multiple thermocouples buried in the dust layer, steady state temperature distribution of the dust layer is found. The parameters are estimated by comparing the experimentally obtained temperature distribution to the numerically predicted temperature distribution

#### 4.1. Estimation of thermal conductivity

Effective thermal conductivity of the dust layer can be estimated by assuming that heat generation at low temperature is negligible. Since the heat generation rate shows significant dependency on the local dust layer temperature, it may not be negligible near the critical temperature of thermal runaway. However, the assumption can be achieved if the hot plate temperature is sufficiently low.

(a) Neglecting heat generation term in Eq. (1) yields,

$$\frac{\partial^2 T}{\partial x^2} = 0 \tag{6}$$

(b) Integration of Eq. (6) with respect to the distance from the hot surface (*x*) leads to a constant value of dT/dx, which yields linear temperature profile in the dust layer. dT/dx is equal to the difference of *T*s and *T*<sub>p</sub> over the layer thickness.

k



Fig. 6. Temperature profile vs. time for 6.4, 12.7, 19.1, and 25.4 mm thick Pittsburgh seam coal dust layer.

(c) From Eq. (2b), thermal conductivity can be written as,

$$x = -h_{\rm t}(T_{\rm s} - T_{\rm a})\frac{{\rm d}x}{{\rm d}T} = \frac{-h_{\rm t}(T_{\rm s} - T_{\rm a}){\rm d}x}{(T_{\rm s} - T_{\rm p})}$$
(7)

where dx = the thickness of the dust layer (0.0127 m).

- (d) In Eq. (7),  $T_s$  can be obtained from the linear extrapolation of experimentally obtained steady state temperature profile of dust layer.
- (e) In Eq. (7), effective total heat transfer coefficient,  $h_t$ , is comprised of convective heat transfer coefficient,  $h_c$ , and radiant heat transfer coefficient,  $h_r$ .

Convective heat transfer coefficient,  $h_c$ , is estimated from the correlation for natural convection of a horizontal plate with hot surface facing up [31,32].

$$h_{\rm c} = \frac{0.54 Ra^{0.25} k_{\rm a}}{L}, \quad \text{for } 10^5 \approx < {\rm Ra} \approx < 10^7$$
 (8)

where  $k_a$  = thermal conductivity of air (W/mK), L = characteristic length as the side of square having the same area of dust layer surface (m), Ra =  $g\beta(T_s - T_a)L^3/\nu\alpha$ , g=gravity (9.81 m/s<sup>2</sup>),  $\beta$  = inverse film temperature (1/K),  $\nu$  = kinematic viscosity of air (m<sup>2</sup>/s),  $\alpha$  = thermal diffusivity of air (m<sup>2</sup>/s).  $T_a$  = ambient temperature (K). All the properties are estimated at the film temperature, the average value of the top surface temperature and ambient temperature. Radiant heat transfer coefficient,  $h_r$ , is obtained by linearizing the radiant heat transfer at the surface as shown in Eq. (9).

$$\dot{q''} = \varepsilon \sigma (T_s^4 - T_a^4) = h_r (T_s - T_a)$$

$$h_r = \varepsilon \sigma (T_s^2 + T_a^2) (T_s + T_a)$$
(9)

where  $\varepsilon$  = the coal emissivity (=0.9 [33]), and  $\sigma$  = Stefan–Boltzmann constant (5.67 × 10<sup>-8</sup> W/m<sup>2</sup> K<sup>4</sup>).

Temperatures measured at heights of 4, 7 and 10 mm in the 12.7 mm thick dust layer exposed to a hot surface temperature of 50 °C are shown in Fig. 7(A). Averaged temperatures from 1000 to 2500 s at each location are deemed to be steady state temperatures: 46, 41, and 37.4 °C for 4, 7, and 10 mm, respectively. Linear extrapolation from the three temperature yields  $T_s$  of 34.5 °C as shown in Fig. 7(B).  $h_c$  of 4.6 W/m<sup>2</sup> K is obtained with  $T_s$  = 34.5 °C, and Ra = 7.55 × 10<sup>5</sup> which satisfies the condition for Eq. (8). In addition,  $k_a$  = 0.026 W/m K, and L = 0.09 m. The radiative heat transfer coefficient,  $h_r$  equals 5.6 W/m<sup>2</sup> K and is estimated with  $T_s$  = 34.5 °C. The total heat transfer coefficient,  $h_t$  = 10.2 W/m<sup>2</sup> K from the summation of  $h_c$  and  $h_r$ .

From Eq. (7), the estimated thermal conductivity is 0.1 W/m K with  $T_s$  of 34.5 °C and  $h_t$  of 10.2 W/m<sup>2</sup> K. The value of the thermal conductivity (0.1 W/m K) is the same as found by Reddy et al. [28] and Jones [10]. Thermal conductivity is assumed to be a constant value, i.e. not a function of temperature, and this is reasonable approximation for Pittsburgh seam coal dust. Negligible difference of thermal conductivity in the range of 50–250 °C was reported by Singer and Tye [26].



Fig. 7. 12.7 mm dust layer temperatures at different heights exposed to a hot surface of 50 °C (A), and steady state temperature distribution in the dust layer and its linear trend line (B).

#### 4.2. Estimation of E and QA

Activation energy (E), and the coupled pre-exponential term (QA) are obtained from a single 12.7 mm thick dust layer experiment using a numerical optimization method. E and QA are estimated by comparing the two steady state dust layer temperature profiles which are, respectively, obtained from the experiment and from the numerical model. E, as an input parameter to the numerical model, is chosen from the best fitting case at which two temperature profiles show the least difference.

The numerical model for steady state temperature profile with the heat generation term being accounted for is developed. Discretization of Eq. (1) with the second order accuracy yields,

$$\frac{T_{i+1} - 2T_i + T_{i-1}}{dx^2} = -\frac{\rho QA}{k} e^{-E/RT_i}$$
(10)

Due to the non-linear term in the right hand side of Eq. (10), an iterative method is required to find converged values of temperature.  $0.1 \,^{\circ}$ C is used as the convergence criterion. The numerical model requires the values of unknown parameters, *E* and *QA* in the heat generation rate term, and the application of bottom and top boundary condition to be a numerically well-posed problem.

The values of *E* and *QA* are required, but neither value is known. Numerous combinations of *E* and *QA* which satisfy the experimental results of steady state at  $T_p$  of 210 °C and of thermal runaway at  $T_p$  of 215 °C are possible. However, a specific relationship between *E* and *QA* exists at the critical hot surface temperature. By correlating *QA* as a function of *E*, the numerical model can be completed with only one unknown parameter, *E*. The existence of the specific correlation between *E* and *QA* can be found from the fundamental steady state thermal runaway theory.

For a given *E* and  $T_p$  at which steady state temperature profile is obtained, as *QA* decreases, temperature distribution consequently becomes closer to the temperature distribution of an inert substance, which implies heat generation rate is an increasing function of *QA*. If *QA* increases, the dust layer temperature increases, but only up to a specific value beyond which heat generation rate term becomes too large for steady state dust layer temperature. Therefore, at critical hot surface temperature where the maximum heat generation rate for steady state is recorded, the value of *QA* becomes also the maximum for a given *E*.

The procedure to obtain a correlation between *E* and *QA* is given below:

(a) The critical non-dimensional heat generation rate term,  $\delta_c$ , is derived by substituting  $T_p$  with  $T_{p3}$  in Eq. (3c). Taking the loga-

rithm of both sides of  $\delta_c$  yields,

$$\ln(QA) = \frac{1}{RT_{p3}}E + \ln\left(\frac{\delta_c RT_{p3}^2 k}{Er^2 \rho}\right).$$
(11)

A correlation that satisfies Eq. (11) for various pairs of *E* and  $\delta_c$  values can be obtained by interpolating the various points of  $\ln(QA)$  vs. *E*.

(b) In Eq. (11),  $T_{p3}$  and  $\delta_c$  are required to calculate the value of  $\ln(QA)$  for a given *E*. For Pittsburgh seam coal used in this study,  $T_{p3}$  ranges from 210 to 215 °C at a layer thickness of 12.7 mm.  $\delta_c$  is calculated from Eq. (5) in which *Bi* and  $\theta_a$  are required. *Bi* is calculated from Eq. (12) derived by substituting *k* with Eq. (7) from its definition [19],

$$Bi = \frac{h_{\rm t}r}{k} = \frac{T_{\rm p} - T_{\rm s}}{2(T_{\rm s} - T_{\rm a})}$$
(12)

In Eq. (12),  $T_s$  is calculated assuming the dust layer as an inert material which yields the linear temperature profile in the dust layer [19]. In Eq. (7), with *k* as 0.1 W/m K, the unknown parameters are  $T_s$  and  $h_t$ . Since  $T_s$  is required for the calculation of  $h_t$ consisting of  $h_c$  and  $h_r$ ,  $T_s$  is initially assumed and updated iteratively to satisfy both Eq. (7) and  $h_t$ .  $\theta_a$  is obtained from Eq. (3b) with the given *E*. Since the correlation between  $\ln(QA)$  and *E* is sought at the criticality,  $T_p$  is replaced by  $T_{p3}$  in the calculation of *Bi* and  $\theta_a$ . Considering 210 and 215 °C as two bounding values of  $T_{p3}$ , two  $\ln(QA)$  and *E* correlations are obtained.

(c) All the parameters required for  $\ln(QA)$  are calculated in step (b) for a given *E*. Now, by varying *E* within the expected range, the value of  $\ln(QA)$  can be obtained for the corresponding *E*. The  $\ln(QA)$  and *E* correlation is obtained by interpolating various points of  $\ln(QA)$  vs. *E*.

An initial value of *E* needed for the calculation of  $\delta_c$  in step (b), is obtained from Table 1, where values of *E* range between 40 to 120 kJ/mol. Taking *E* of 120 kJ/mol and  $T_{p3}$  of 210 °C as a sample case,  $\delta_c$  of 8.78 is calculated using Eq. (5) with  $\theta_a$  as –11.6 and *Bi* as 0.9.  $T_s$  of 89 °C and  $h_t$  of 14.2 W/m<sup>2</sup> K are calculated. ln(*QA*) of 36.4 is calculated for *E* of 120 kJ/mol.

For the same *E*, with  $T_{p3}$  of 215 °C which yields  $T_s$  of 90.4 °C and  $h_t$  of 14.3 W/m<sup>2</sup> K, ln(*QA*) becomes 36.1 with  $\delta_c$  of 8.93.  $\theta_a$  of –11.7 and *Bi* of 0.9 are calculated. Two values of ln(*QA*) are derived from Eq. (11) for 120 kJ/mol: 36.4 with  $T_{p3}$  of 210 °C and 36.1 with  $T_{p3}$  of 215 °C.

In the same way, decreasing *E* to 40 kJ/mol, corresponding values of *QA* can be obtained for each  $T_{p3}$ . Interpolating various points of ln(*QA*) and *E*, two linear correlations with *R*-square of unity are obtained as shown in Fig. 8(A): ln(*QA*) = 2.5777 × 10<sup>-4</sup>*E* + 5.4685



**Fig. 8.** Pairs of *E* and  $\ln(QA)$  with  $T_p$  of 210 and 215 °C (A) and numerical estimation at  $T_p$  of 210 °C with *E* = 40 and 120 kJ/mol and corresponding *QA*s obtained from the correlation with  $T_{p3}$  of 215 °C (B).

with  $T_{p3}$  of 210 °C, and  $\ln(QA) = 2.5524 \times 10^{-4}E + 5.5029$  with  $T_{p3}$  of 215 °C. Nugroho et al. [30] also reported linear relationship between  $\ln(QA)$  and *E* at criticality for various coal types.

Now, *QA* in the heat generation rate term in Eq. (10) can be replaced by a function of *E* which leaves *E* as the only unknown parameter such that experimental results can be compared to the numerical results by changing only *E*. It also provides the lower and upper bounding values of *QA* for a given *E* which satisfies the experimental fact that the maximum hot surface temperature for steady state lies between 210 and 215 °C.

Lower and upper surface boundary conditions need to be applied to fulfill the well-posed problem. The lower surface boundary condition of Eq. (2a) is applied to the model as an input value. The upper surface boundary condition of Eq. (2b) requires k,  $T_s$ , and  $h_t$ .  $T_s$  and  $h_t$  can be obtained from Eq. (7) as can be in the calculation for the ln(QA) and E correlations if  $T_p$  is known. Therefore, the top boundary condition is defined by the bottom boundary condition, which leaves  $T_p$  as the only input value in terms of the application of boundary conditions to the numerical model.

The numerical model for steady state dust layer temperature profile is completed with the input values of  $T_p$  and E. By varying E, the steady state dust layer temperature profile obtained from the numerical model can be compared to the experimentally obtained temperature profile at the same  $T_p$ .  $T_p$  can be any value as long as steady state temperature distribution is obtained, but the effect of E and QA is the most distinctive on the dust layer temperature profile, if  $T_p$  is closer to the  $T_{p3}$ . Since the steady state temperature distribution at 210 °C is available from the experiment, and in the range of  $T_{p3}$ ,  $T_p$  of 210 °C is used for comparison.

Average dust layer temperatures from 5000 to 7000s at 4, 7, and 10 mm heights are used as steady state temperatures: 188.6 °C at 4 mm, 157.5 °C at 7 mm, and 119.6 °C at 10 mm as can be seen in Fig. 6(B). This experimentally obtained temperature distribution of 12.7 mm thick dust layer is compared to the numerically estimated temperature distributions with two bounding values of E, 40 and 120 kJ/mol as found in Table 1, and with the ln(QA) and E correlation derived from  $T_{p3}$  at 215 °C in Fig. 8(B).  $T_{p3}$  of 215 °C is not the actual hot surface temperature to which dust layer is exposed, but the temperature used to obtain the ln(QA) and E correlation. Less temperature rise in the dust layer is found with higher activation energy in the numerical model. Considering activation energy as thermal barrier for exothermic reaction to occur, less heat is generated in the dust layer with higher activation energy for the same  $T_p$ such that less temperature rise is reasonable. Reasonable temperature distributions are estimated from the numerical model based on the bounding values of *E* as shown in Fig. 8(B).

The sums of absolute values of temperature differences at 4, 7, and 10 mm locations between numerical estimation and experimental data at various activation energies are shown in Fig. 9(A). The best fitting curve of the numerical model to the experiment yields two optimized values of *E* (and *QA*) from each  $\ln(QA)$  and *E* correlation: one based on  $T_{p3}$  of 210 °C and the other one based on  $T_{p3}$  of 215 °C, specifically considering the steady state temperature of 12.7 mm thick dust layer exposed to  $T_p$  of 210 °C. With the least temperature difference between numerical model and experimental data at 4, 7, and 10 mm as a criterion, *E* of 83.1 kJ/mol (*QA* of  $4.8 \times 10^{11}$  J/kg s), and *E* of 61.7 kJ/mol (*QA* of  $1.7 \times 10^9$  J/kg s) are obtained

Numerically estimated temperature profiles with the obtained best fitting values of E (and QA) are plotted in Fig. 9(B) showing relatively good agreement with the experimental data except for the 10 mm height. This may be due to the higher convective and radiant heat loss near the top surface in actual conditions. Since the dust layer surface is composed of particles, and thus, the real surface area on which the amount of convective and radiant heat loss depends is larger than the theoretical model for numerical estimation.

#### 4.3. Estimation of $T_{p3}$ for other thicknesses

From section 4.2, *E* and *QA* are estimated from a single 12.7 mm thick dust layer with multiple thermocouples in it. *E* is expected to range from 61.7 to 83.1 kJ/mol with *QA* of  $1.7 \times 10^9$  and  $4.8 \times 10^{11}$  J/kg s, correspondingly. The estimated *E* and *QA* values are used to predict the critical hot surface temperatures ( $T_{p3}$ ) for the other thicknesses: 6.4, 19.1, and 25.4 mm. The estimation of  $T_{p3}$  for a given dust layer thickness is practically useful, since in reality, dust layer thickness is measurable, and thus, the maximum safe hot surface temperature for thermal runaway often needs to be known. In this section, the predicted  $T_{p3}$  for each thickness is compared to the experimentally obtained  $T_{p3}$  for the purpose of validation of the current numerical estimation method of *E* and *QA*. The values of *E* and *QA* are also compared to the values obtained from widely known multiple tests method. Detailed calculations of multiple tests method are included in Appendix A.

 $T_{p3}$  is obtained from the case that  $\delta$  from Eq. (3c) with  $T_p$  replaced by  $T_{p3}$  becomes the same with  $\delta_c$  from Eq. (5). Since Bi in Eq. (5) includes a dependent variable,  $T_s$ , which is determined from  $T_{p3}$ with the assumption of the dust layer being an inert material,  $T_{p3}$ should be initially assumed and improved iteratively until it satisfies both values of  $\delta_c$  from Eqs. (3c) and (5). Calculation step for  $T_s$ is the same with step (b) in Section 4.2.



**Fig. 9.** Sum of the absolute values of temperature differences between numerical estimation and experimental data in 4, 7, and 10 mm heights at various activation energies (A) and numerically estimated dust layer temperature distribution with obtained *E*: 61.7 and 83.1 kJ/mol (B).

Since the values of *E* and *QA* are not a specific single value, but a range, mean values of *E* and *OA* are selected for the prediction of  $T_{p3}$  of the other thicknesses: 6.4, 19.1, and 25.4 mm. To check the prediction uncertainty of the numerical model, two bounding pairs of values of *E* and *QA* are also used. *E* of 72.4 kJ/mol is calculated as the mean value, and QA of  $2.8 \times 10^{10}$  J/kg s is obtained by averaging the values of ln(QA) from the two ln(QA) and E correlations with *E* of 72.4 kJ/mol. The mean values of *E* and *QA* correspond to the case that  $T_{p3}$  is assumed to be the mean value of 210 and 215 °C for 12.7 mm thick dust layer. Two bounding pairs of E and QA were estimated from the fact that  $T_{p3}$  lies between 210 and 215 °C for 12.7 mm thick dust layer. The associated parameter values and calculated  $T_{p3}$  values with the three pairs of *E* and *QA* are shown in Table 2. Relative deviations of estimated  $T_{p3}$  for each thickness and for each pair of E and QA are shown in Fig. 10 in comparison with the experimental values. Since  $T_{p3}$  has a 10 °C error range based on the test resolution of  $T_p$  in the experiment, if the predicted  $T_{p3}$  for the other thicknesses are located in the gray box area in Fig. 10, the current estimation procedure of E and QA can be considered reasonable, and if not, the magnitude of the deviation implies the error range of the current method.

For the mean values of *E* and *QA*, the predicted  $T_{p3}$  for the other thicknesses are located in the experimental range (gray box) which indicates that the use of mean values of *E* and *QA* are reasonable. An additional implication is that the real  $T_{p3}$  for 12.7 mm thick dust layer is closer to 212.5 °C. For the bounding pairs of values of *E* and *QA*, the predicted  $T_{p3}$  for 19.1 mm thick dust layer falls within the experimental range, however for 6.4 mm and 25.4 mm thick dust layers, predicted  $T_{p3}$  values fall outside of the experimental range by a maximum of 5.4 and 3.8 °C, respectively. The deviation may be caused by various approximations used for the calculations such

Table 2

Estimated  $T_{p3}$  and associated parameter values for the other thicknesses: 6.4, 19.1, and 25.4 mm from the mean and bounding values of *E* and *QA*.

E (kJ/mol)	QA (J/kgs)	2 <i>r</i> (mm)	$T_{s}$ (°C)	Bi	$\theta_{a}$	$\delta_{\rm c}$	$T_{p3}$ (°C)
72.4	$\textbf{2.8}\times10^{10}$	6.4 19.1 25.4	130.6 71.1 60.2	0.5 1.3 1.6	-7.2 -6.9 -6.7	2.5 4.4 4.7	245.2 194.2 180.8
61.7	$1.7\times10^9$	6.4 19.1 25.4	134.7 70.6 59.2	0.5 1.3 1.6	-6.2 -5.8 -5.7	1.9 3.3 3.6	255.4 192.2 176.2
83.1	$\textbf{4.8}\times\textbf{10}^{11}$	6.4 19.1 25.4	127.1 71.1 60.6	0.5 1.3 1.6	-8.3 -7.9 -7.7	3.0 5.5 6.0	236.8 194.4 183.0



**Fig. 10.** Estimated  $T_{p3}$  for each thickness and deviation of *E* as compared to the experiment.

as the dust layer being an inert material which yields lower  $T_s$  than the actual and subsequent error of Bi, and the approximate solution of  $\delta_c$  of Eq. (5). However, considering that the amount of deviation is not much larger than the experimental error range and that the mean value of E and QA predicts  $T_{p3}$  within the range, the numerical method to estimate E and QA is considered to be stable and reasonable. The deviation can be decreased by finding more narrow range of  $T_{p3}$  for 12.7 mm thick dust layer from which the  $\ln(QA)$  and Ecorrelations are derived.

#### 5. Conclusion

Whether or not a hot surface temperature is sufficiently high to cause thermal runaway is determined by the comparison of the non-dimensional heat generation rate ( $\delta$ ) with the critical heat generation rate ( $\delta_c$ ). If  $\delta$  is less than  $\delta_c$ , meaning heat generation rate at the current hot surface temperature is less than the critical temperature, thermal runaway is not expected to occur. For this comparison, not only are measurable physical properties, such as density, ambient temperature, and thickness of dust layer, required, but also thermal and kinetic parameters, including thermal conductivity (k), activation energy (E), and coupled pre-exponential term (QA) need to be known.

## 154 **Table 3**

The minimum and maximum values of activation energy (E) and the corresponding couple	d pre-exponential term (QA) with the combination of LITs
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Layer thickness (mm)	6.4	19.1	25.4	E(kJ/mol)	QA (J/kg s)
The possible maximum steady state hot surface temperature $(T_{p3})$ (°C)	250 240	190 200	180 190	66.8 89.1	$\begin{array}{c} 6.6\times10^9\\ 1.8\times10^{12} \end{array}$

Exploring thermal runaway theory, hot surface ignition tests have been conducted with four different dust layer thicknesses: 6.4, 12.7, 19.1 and 25.4 mm using Pittsburgh seam coal dust and a slightly modified standard ASTM E 2021 test apparatus. The ranges of the maximum hot surface temperature ( $T_{\rm P3}$ ) for the dust layer to remain in steady state were found for each thickness. Multiple thermocouples were located in the dust layer such that the steady state temperature distribution was determined at a given hot surface temperature.

k, E and QA were obtained from a single 12.7 mm thick dust layer with a numerical optimization method. k of 0.1 W/mK was estimated assuming negligible heat generation at the hot surface temperature of 50 °C, and matched the previously reported literature value [10,25]. To estimate E and QA, the multiple tests method requires different hot plate temperatures, and thus, different thicknesses of dust layer are needed to be tested. Instead of multiple hot surface temperatures, multiple thermocouples enabled the comparison of steady state dust layer temperature profiles obtained from a numerical optimization method and the experiment to be used to estimate E and QA. E of 61.7 to 83.1 kJ/mol and QA of  $1.7 \times 10^9$ – $4.8 \times 10^{11}$  J/kg s were obtained from a single 12.7 mm thick dust layer. The estimated E and QA were then used to predict the critical hot surface temperatures of the other thicknesses: 6.4, 19.1, and 25.4 mm for validation purpose. From the experiment, the critical hot surface temperatures for 6.4, 19.1, and 25.4 mm thick dust layer ranged between 240 and 250 °C, 190 and 200 °C, and 180 and 190°C, respectively, with 10°C experimental error range. The mean values of E and QA, E of 72.4 kJ/mol and QA of  $2.8 \times 10^{10}$  J/kg s, predicted the critical hot surface temperatures for the other thicknesses within the experimental error range and the maximum deviation outside the experimental error range of  $6 \circ C$  was observed with two bounding values of E and QA, E of 61.7 kJ/mol (and QA of  $1.7 \times 10^9$ J/kgs) and 83.1 kJ/mol (and QA of  $4.8 \times 10^{11}$  J/kg s), which shows the presented numerical optimization method to estimate E and QA to be reasonable. The estimated E and QA values also matched well with the values obtained from widely accepted multiple tests method (E of 66.8-89.1 kJ/mol and QA of  $6.6 \times 10^9 - 1.8 \times 10^{12}$  J/kg s) as described in Appendix A. The current estimation method with Pittsburgh seam coal shows satisfactory results as it applies to different dust layer thicknesses, and as compared to the multiple tests method. Further research must address broader applications of this method for different coal types.

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#### Appendix A

Logarithm of the critical non-dimensional heat generation rate term,  $\delta_c$ , from Eq. (3c) substituting  $T_p$  with  $T_{p3}$ , yields,

$$\ln\left(\frac{\delta_{\rm c}T_{\rm p3}^2}{r^2}\right) = -\frac{E}{RT_{\rm p3}} + \ln\left(\frac{E}{R}\frac{\rho QA}{k}\right) \tag{13}$$



**Fig. 11.** Possible layer ignition temperatures (LITs) data of Pittsburgh seam coal dust layer to obtain *E* and *QA*.

Activation energy (*E*) is obtained from the slope of the linearized interpolation line of the multiple points of  $\ln(\delta_c T_{p3}^2/r^2)$  vs.  $1/T_{p3}$ . Several different  $T_{p3}$  values which can be obtained from the experiments with different dust layer thicknesses are required for plotting multiple points. With this method, Reddy et al. [28] reported *E* and *QA* of Pittsburgh seam coal dust as shown in Table 1 based on the results of Mirron and Lazzara's tests [8].

To have an independent check of *E* numerically estimated from 12.7 mm thick dust layer, test results from 6.4, 19.1, and 25.4 mm thickness are used for the calculation of multiple tests method. LITs of 250, 200, and 190 °C were obtained for 6.4, 19.1, and 25.4 mm thick Pittsburgh coal dust layer, and thermal runaway was not observed at 240, 190, and 180 °C for each thickness. Due to the relatively large test resolution of hot surface temperature which is 10 °C,  $T_{p3}$  for each thickness can be any temperature between the two boundary temperatures. For example, any temperature greater than or equal to 240 °C and less than 250 °C can be  $T_{p3}$  of 6.4 mm thick layer, and consequently, the set of  $T_{p3}$  of each thickness can vary as much leading to different value of *E*. Jones et al. [34] also mentioned the error in *E* estimation is mainly caused by the uncertainty of  $T_{p3}$ .

From reviewing all the possible cases of  $T_{p3}$  for each thickness, the combinations of  $T_{p3}$  resulting in the minimum and maximum values of *E* and corresponding *QA*'s are shown in Table 3. As shown in Table 3, 250 °C is used as  $T_{p3}$  for 6.4 mm thickness at which thermal runaway was observed. However, adopting infinitesimally small test resolution, the possible  $T_{p3}$  for steady state can become 250 °C with negligible error. Estimated *E* ranges from 66.8 to 89.1 kJ/mol. Fig. 11 shows relative relationship of the two data sets from which the minimum and maximum value of *E* results.

## Table 4

E and Q	<u>A</u> from	numerical	estimation	and n	nultipl	e tests	method
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	Numerical estimation	Multiple tests method		
E (kJ/mol)	61.7-83.1	66.8-89.1		
QA (J/kg s)	$1.7 \times 10^9  4.8 \times 10^{11}$	$6.6\times 10^9  1.8\times 10^{12}$		

Table 4 shows activation energy and pre-exponential term obtained from numerical estimation and the multiple tests method. *E* from numerical estimation is slight less than multiple tests method, but shows good agreement.

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